

# Update on Advection-Diffusion Purge Flow Model

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**Lubos Brieda**

Particle In Cell Consulting LLC



# What is purge?

- Purging is the process of continuously flowing clean gas into a sensitive electronic or optical equipment to maintain internal cleanliness
  - Typically Grade B or Grade C gaseous nitrogen is used
- Common on spacecraft instruments:
  - Used on MMS and GOES-R missions
  - Both missions utilize microchannel plates (FPI on MMS and MPS-LO on GOES-R), highly sensitive to molecular contamination
  - GOES-R also contains UV (SUVI) and optical (ABI and GLM) instruments which are also sensitive to molecular contamination
- The working assumption is gas flow “volume exchanges” clean up internal environment, and that positive pressure prevents infiltration of ambient environment
- Purge is also assumed to prevent infiltration of “dust” particulates

# Impact on Design and I&T

- There are also drawbacks to purging:
  - Purge ports must be added to instruments
  - Flight-ready purge plumbing must be installed on the spacecraft if T-0 purge used
  - Purge can be a contaminant source if not designed right
  - Purge carts must be designed and built – often a non-trivial exercise due to use of high-pressure K-bottles
  - Analysis is required to determine necessary purge flow rates and time-off purge
  - Time off purge can become a driving factor in I&T activities
- Purge can also sometimes lead to false sense of security
  - Due to complex internal multi-cavity geometry, purge gas may flow out of the device without passing over the sensitive component

**What purge flow rate is needed and how much time off purge can an instrument tolerate?**

# Analytical Model

- Scialdone introduced a simple model in “Preventing Molecular and Particulate Infiltration in Confined Volume”, *SPIE 2784*, 1999.

- Follow up to 1978 NASA TP-1172, “Water-Vapor Pressure in C.V.”

- Start with mass conservation law

$$V \frac{dP}{dt} = C(P_0 - P) - Q(P - P_u)$$

- Can be obtained from  $\frac{\partial \rho}{\partial t} + \nabla \cdot \vec{u} = 0$  assuming  $\frac{dT}{dt} = 0$

- Integrate using integration factor to yield

$$P = k \exp \left[ -\frac{C + Q}{V} t \right] + \frac{CP_0 + QP_u}{C + Q}$$

- Where  $k$  is found from  $P(0) = P_i$

$$P = \left[ \frac{C(P_i - P_0) + Q(P_i - P_u)}{C + Q} \right] \exp \left( -\frac{C + Q}{V} t \right) + \frac{CP_0 + QP_u}{C + Q}$$

- If  $Q = 0$  (no purge) and  $P_u = 0$  (clean gas), the above reduces to

$$P = (P_i - P_0) \exp \left( -\frac{C}{V} t \right) + P_0$$

- Scialdone also experimentally studied the aperture conductance and found that

$$\tau_v \equiv \frac{V}{C} = 0.42 * 24 * \left( \frac{V}{A} \right) \text{ (but doesn't agree with graph)}$$

$P$  (internal partial pressure of contaminant)

$P_0$  ambient pressure of contaminant

$C$  conductance through aperture

$Q$  flow rate of purge gas

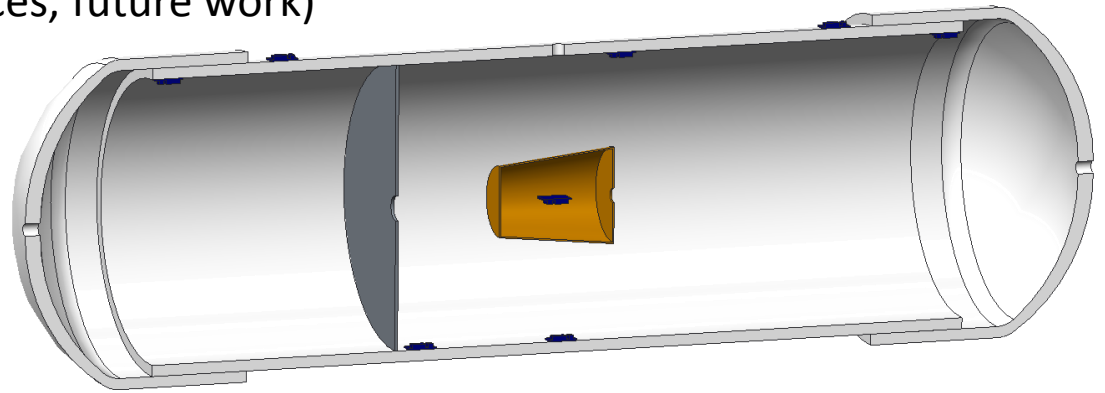
# Research Objectives

- Scialdone's model is easy to use but:
  - The model is 0D, does not take into account details of internal geometry
  - Pressure differential or total pressure not used
  - C/V ratio applicable only to thin apertures
- The objective of this study is to investigate the simple model in more detail:
  - It seems too simple to be true. But maybe it is?

**Simple models can be used by engineers to quickly obtain answers.  
On the other hand, they can result in false findings or an overly simplified  
model**

# Research Summary

- Performed a combined experimental and numerical effort to investigate purging in more detail: **SPIE-2014**
- A cylindrical enclosure purged with GN2
  - Instrumented with multiple miniature humidity and pressure sensors
  - Included internal flow obstructions
  - Axial symmetry desired in order to allow RZ simulation
- Sensors used to measure rate of water infiltration after purge was stopped
- Numerical tools developed to simulate evolution of contaminant density with or without gas flow
  - Combined **advection/diffusion** and **incompressible Navier Stokes** solver
- Main topics:
  1. Spatial and temporal variation
  2. Geometry impact (vortices, future work)
  3. Impact on particulates



# **PART I: EXPERIMENTAL EFFORT**

# Experimental Setup

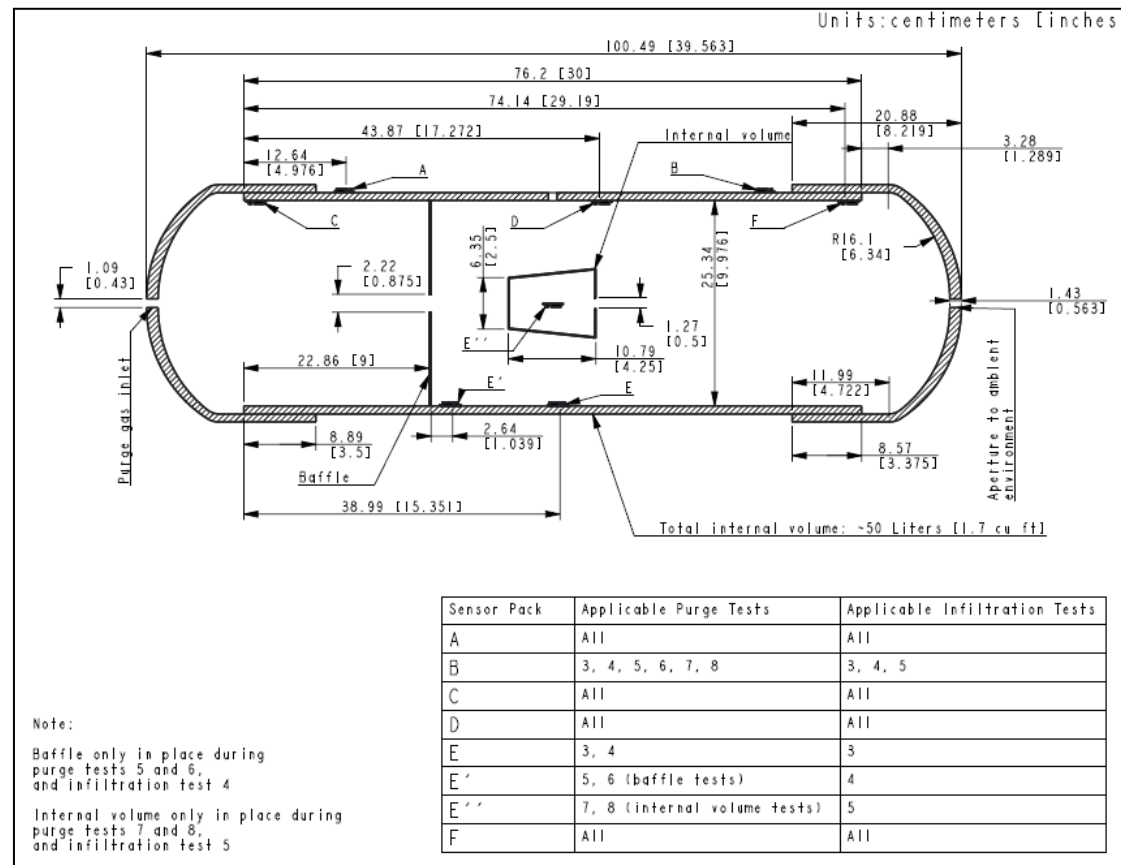
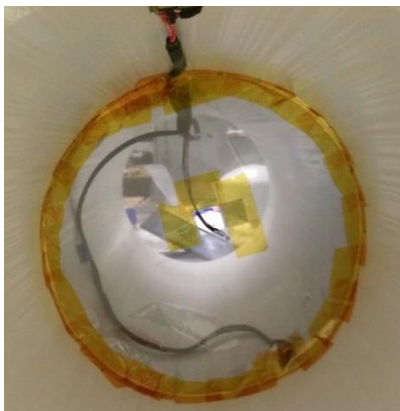
- Experiment consisted of a size 10 PVC pipe capped on both ends with size 10 end caps, 3.3 feet long
- 0.43" diameter holes were drilled in the center of both end caps
  - One was connected to a purge system, consisting of a pressure regulator, and a flow meter
  - The other was left open to the atmosphere
- Purge gas was provided by Grade C K bottle or by using a boil-off dewer house gas
- Testing was performed at GSFC propulsion lab





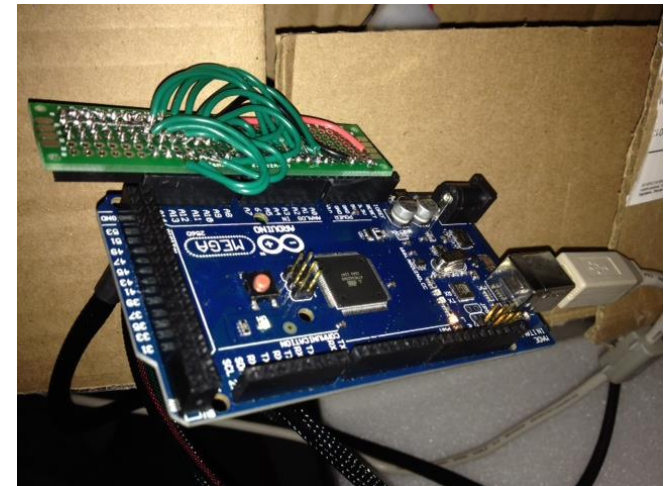
# Internal Setup

- Performed series of tests:
  - Empty internal cavity with 2scfh ( $\sim 1\text{L/min}$ ), 4scfh ( $\sim 2\text{L/min}$ ), and 1scfh ( $\sim 0.5\text{L/min}$ )
  - Baffle @ 2scfh
  - Secondary volume @ 2scfh
- Baffle constructed from ESD bagging
- Internal volume constructed by wrapping a coffee cup in foil



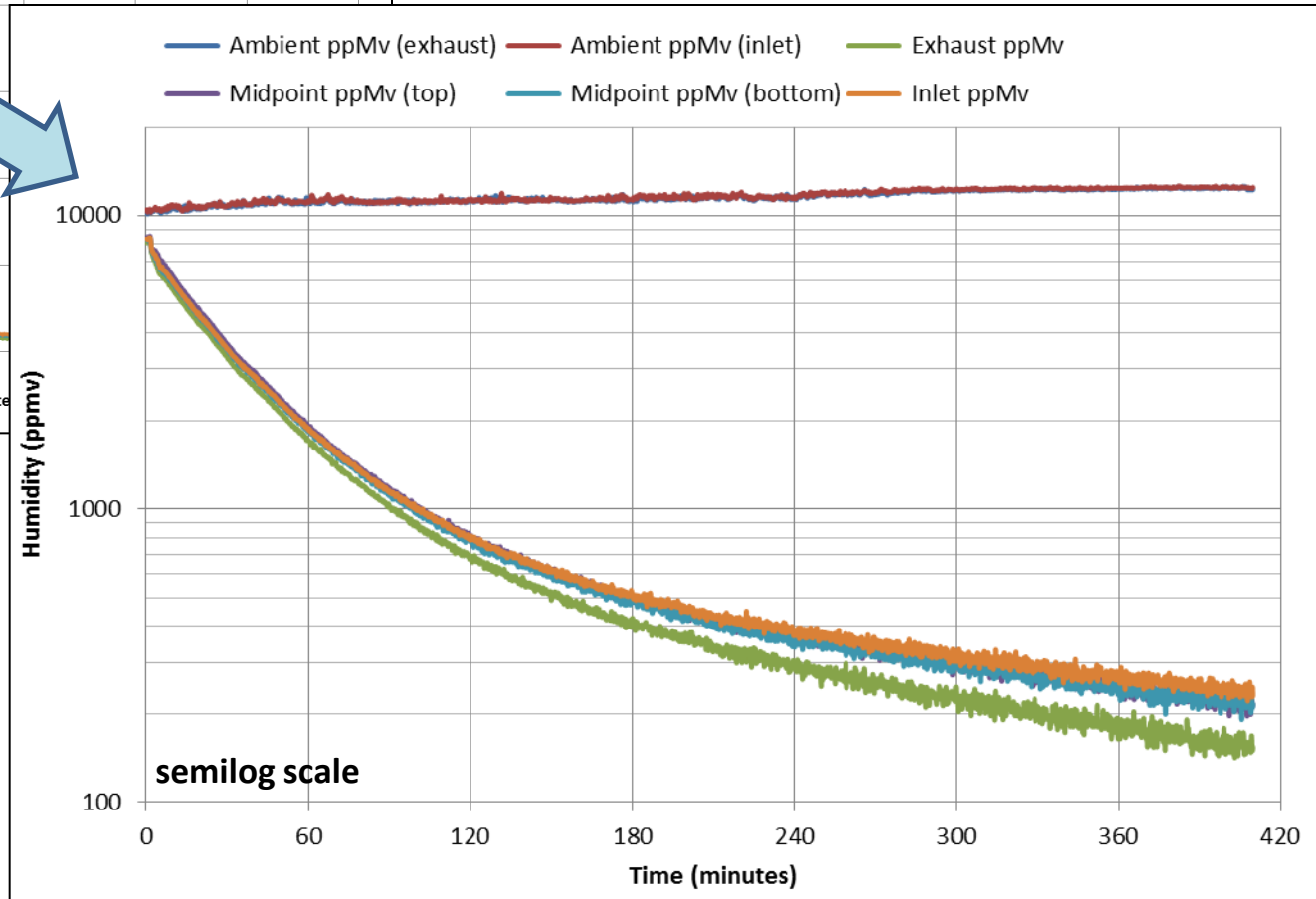
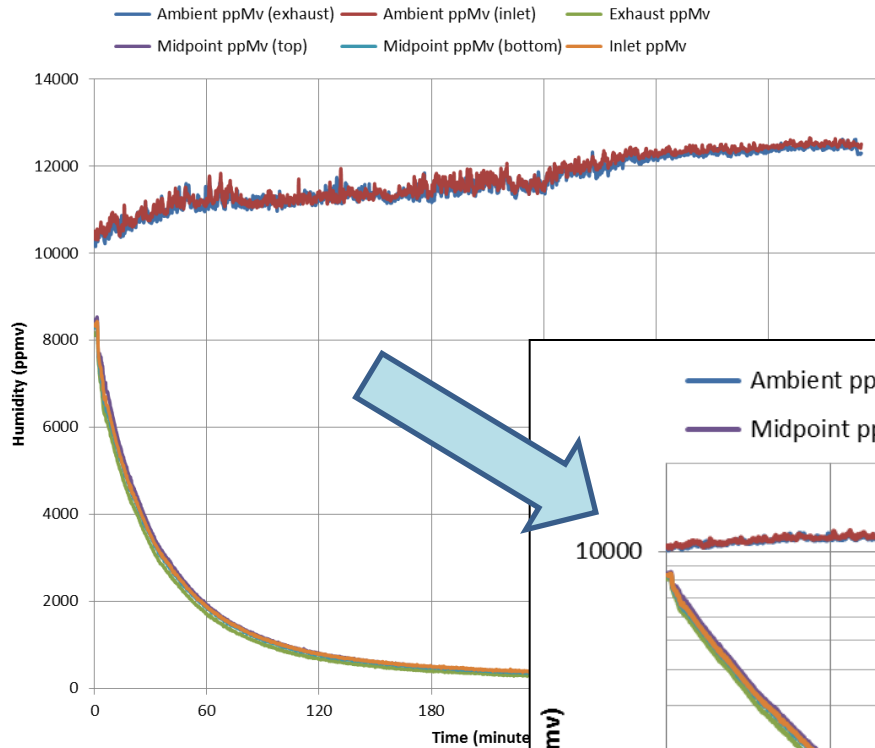
# Sensors and Data Acquisition

- Used Arduino Mega for a data collection
- Sensors:
  - Honeywell HIH-5030 capacitive relative humidity sensor capable of measuring from 0 to 100% RH
  - Freescale Semiconductor MPXHZ6130A piezoresistive transducer with analog ratiometric output used to measure absolute pressure
  - Analog TMP36 sensor used to measure temperature
- Sensors mounted on a breakout board connected to a harness
- Typically six sensors were used: four internal, two external
- Java program to sample sensors every 10 seconds



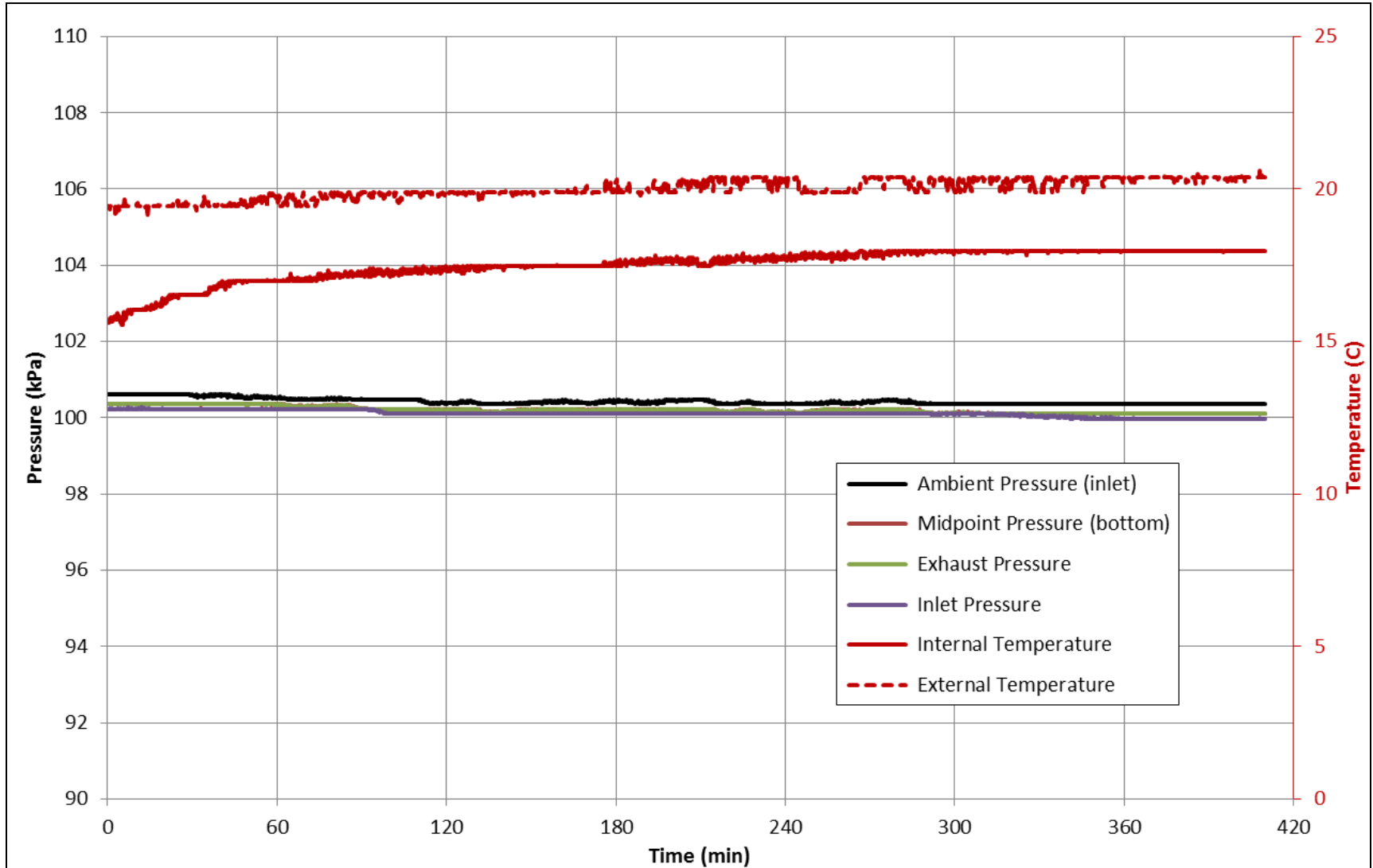
# Typical Results (2 L/min)

- Negligible variation in internal humidity for empty cavity



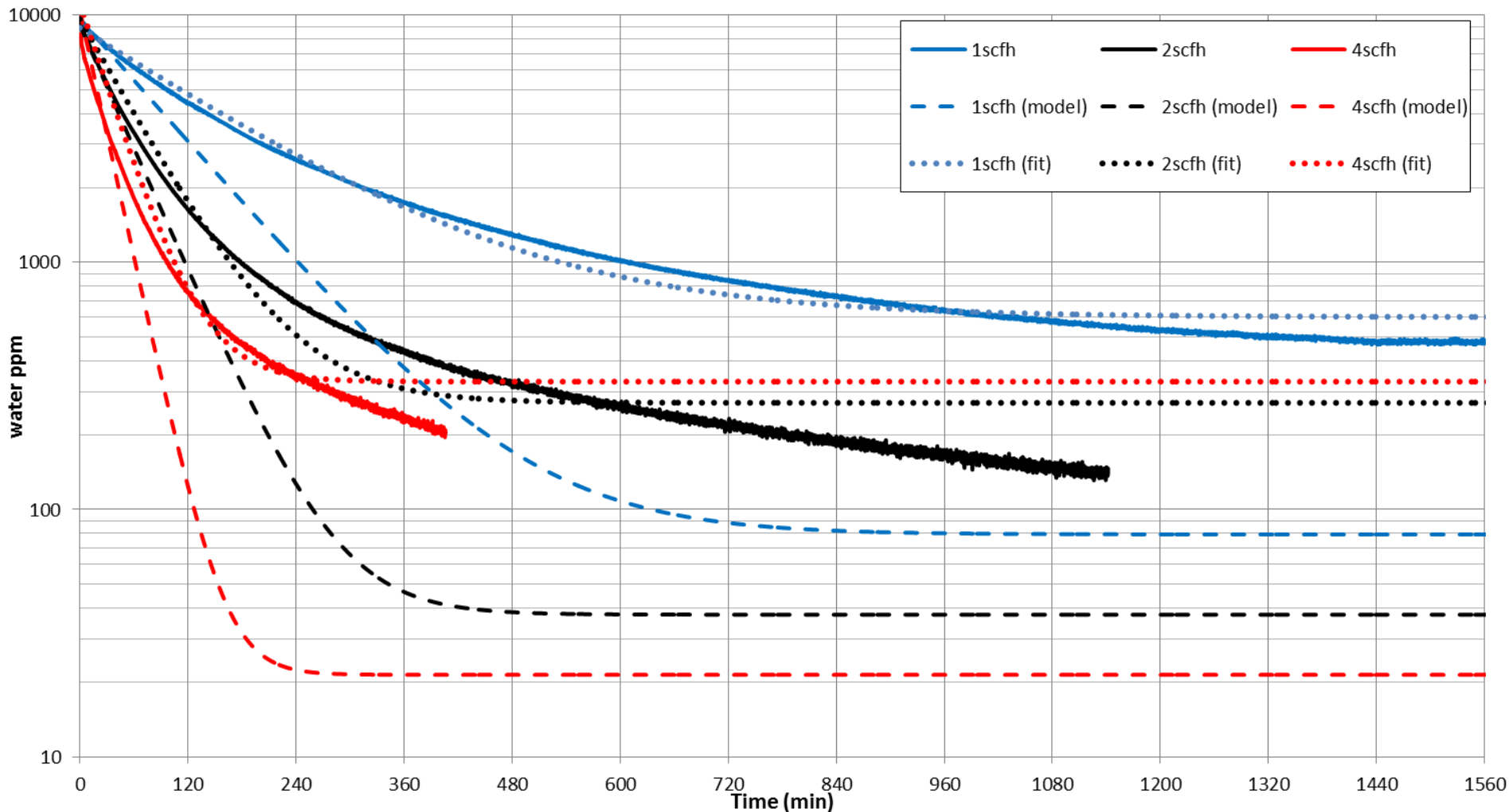
# Temperature and Pressure

- No impact on pressure or temperature from purge activation



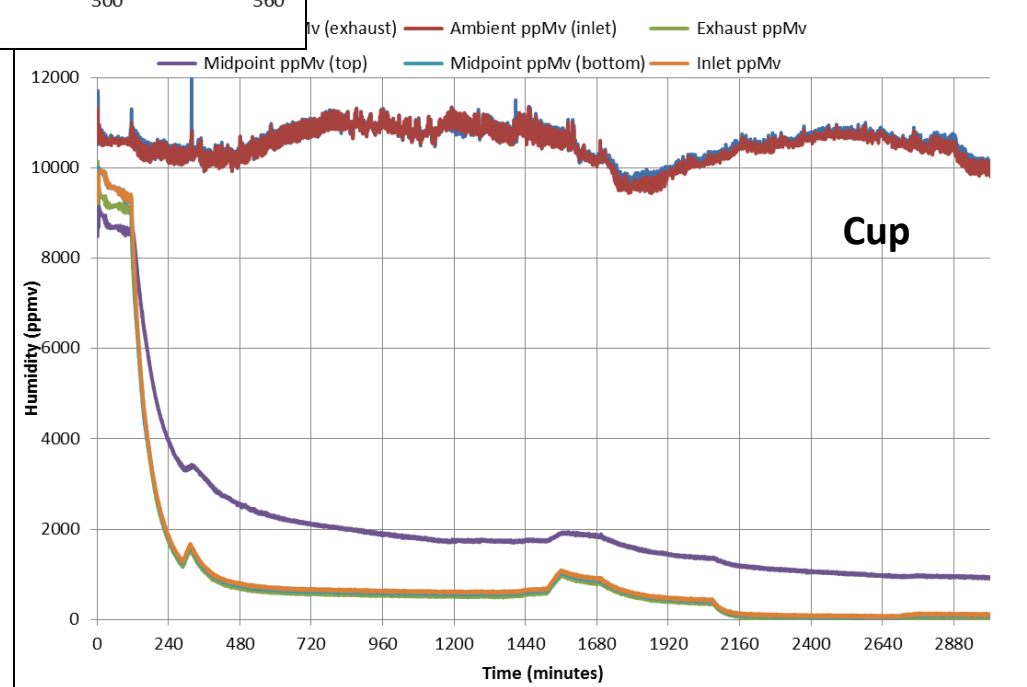
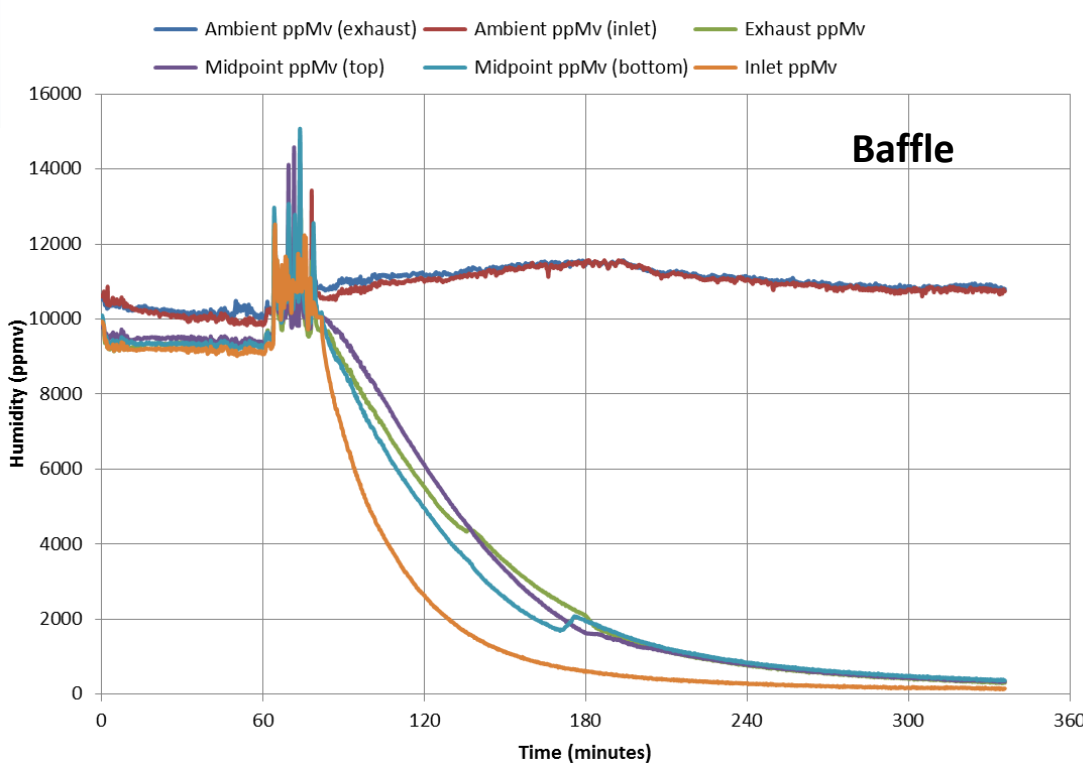
# Comparison with Model

1. Double exponential results in better fit than a single exponential
2. Significant difference with known values for  $Q$ ,  $V$ , and  $C$
3. Concentration continues to drop while model predicts asymptotic behavior



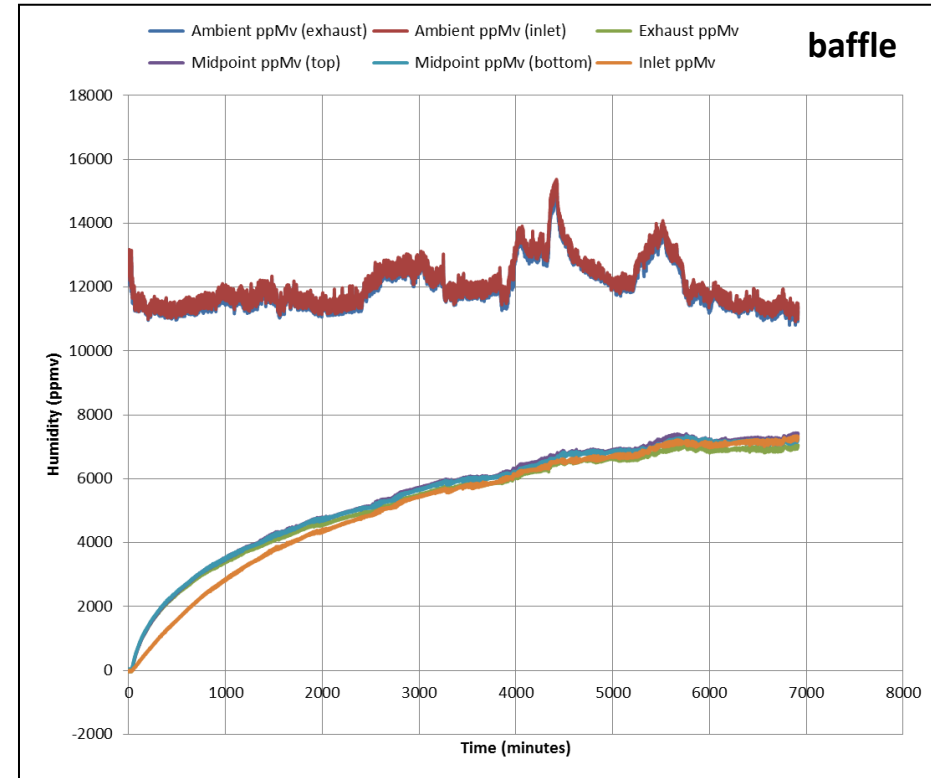
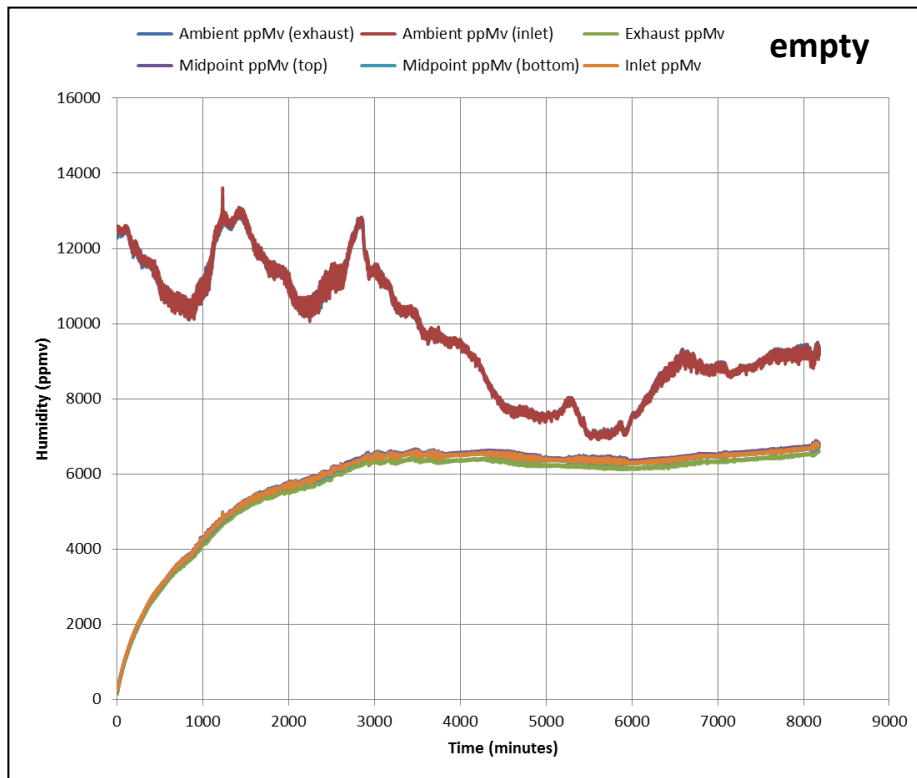
# Internal Variation

- Adding internal obstacles results in a non-uniform internal concentration



# Water Infiltration

- After the completion of each purge run, flow was stopped, and water was allowed to diffuse back into the cavity
- Plots below shows typical response
  - Internal humidity leveled off at approximately 75% of external value after 6 days – **thick orifice effect**
  - Internal sensors reached ambient values when cap was removed



## **PART II: NUMERICAL MODEL**



# Numerical Model

- Java code developed to study flow in more details
- Advection-diffusion equation for water density

$$\frac{\partial n_w}{\partial t} + \nabla \cdot (-D \nabla n_w + \vec{u} n_w) = R$$

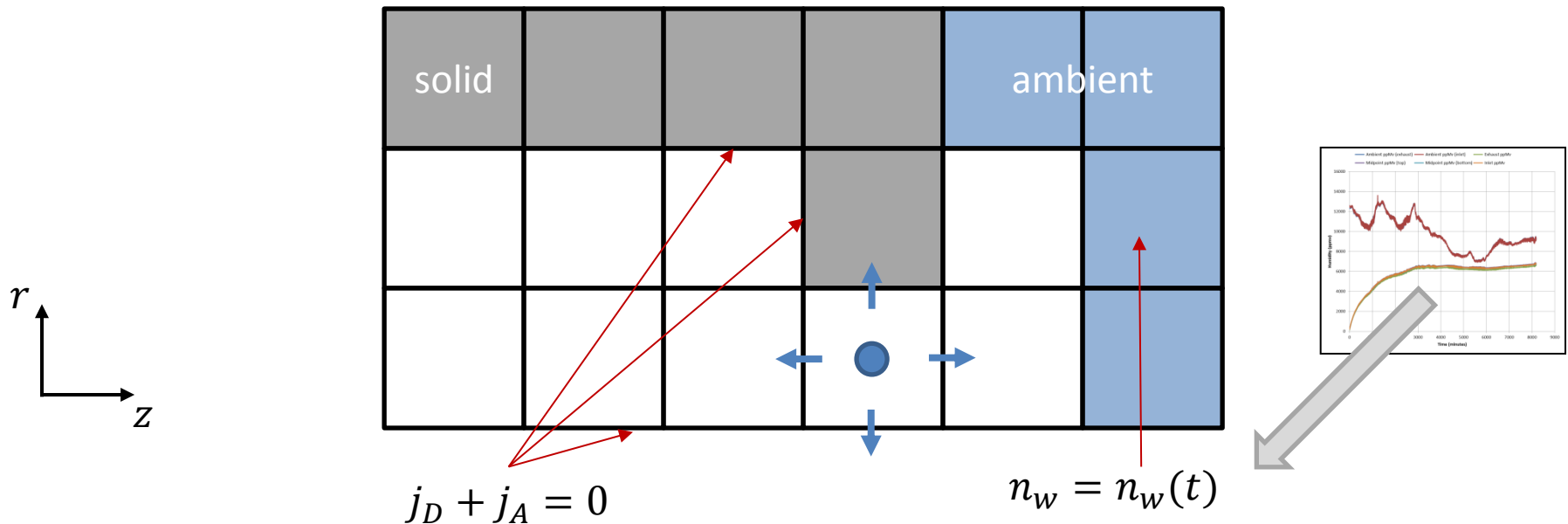
- Incompressible Navier Stokes solver to obtain gas velocity

$$\left( \frac{\partial \vec{u}}{\partial t} + \vec{u} \nabla \cdot \vec{u} \right) = -\frac{1}{\rho} \nabla p + \frac{\mu}{\rho} (\nabla^2 \vec{u})$$

- Solution strategy:
  - Integrate NS equations, independent of water density
  - Use new velocity field to update water density
- Implied assumptions:
  - Water density has no impact on flow solution
  - The only source of water is the ambient environment (for now...)
  - Turbulence not modeled (also for now)

# Advection Diffusion Solver

- Solution obtained on a staggered grid:
  - Water density known at cell centers
  - Velocity known at centroids of cell edges
- Assumed axial symmetry, results in an axisymmetric (RZ) code
- No flux through walls implies  $(D\nabla n_w + n_w \vec{u}_w) \cdot \hat{n} = 0$  along boundaries
  - Also true along axis of rotation,  $r = 0$
- Sugar-cubing used to mark solid cells
- Ambient density varied based on experimental data



# Incompressible NS Solver

- Solved using projection method. Velocity integration split to part due to advection/viscosity, and to part due to pressure

$$\frac{\vec{u}^t - \vec{u}^k}{\Delta t} = -\vec{u} \nabla \cdot \vec{u} + \nu (\nabla^2 \vec{u})$$

$$\frac{\vec{u}^{k+1} - \vec{u}^t}{\Delta t} = -\frac{1}{\rho} \nabla p$$

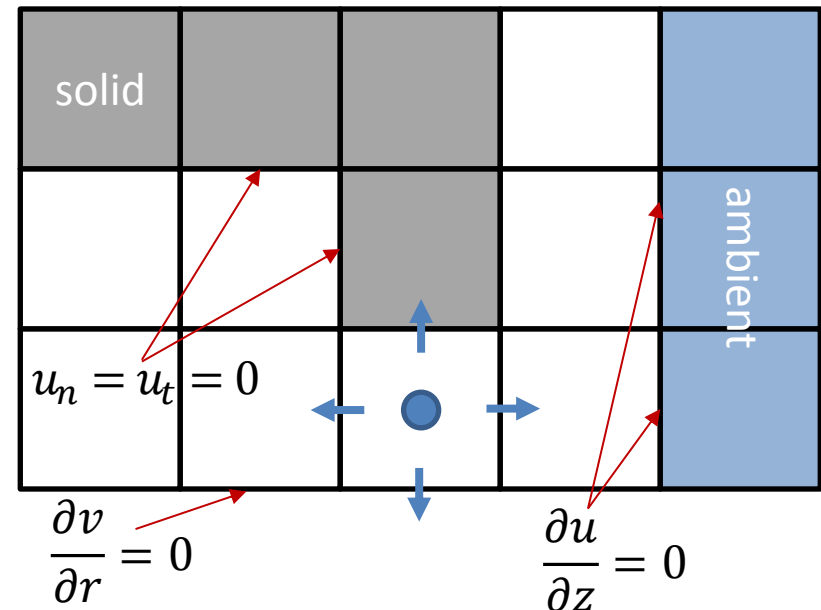
- “Temporary” velocity found from  $\vec{u}^t = \vec{u}^k - \Delta t (\vec{u} \nabla \cdot \vec{u} + \nu \nabla^2 \vec{u})$
- Mass conservation  $\nabla \cdot \vec{u}^{k+1} = 0$  used in equation 2 to obtain for pressure

$$\nabla^2 p = \frac{\rho}{\Delta t} \nabla \cdot \vec{u}^t$$

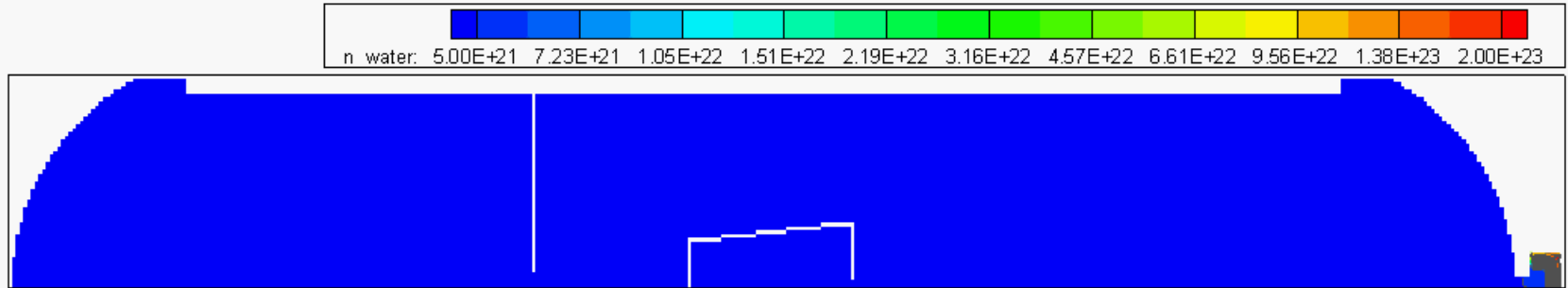
- New velocity then found from

$$\vec{u}^{k+1} = \vec{u}^t - \frac{\Delta t}{\rho} \nabla p$$

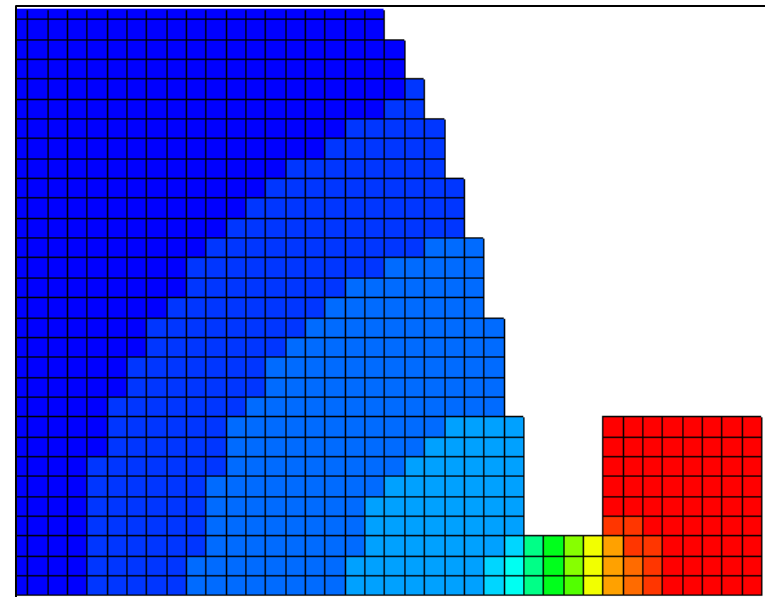
- Same grid as AD solver
  - Pressure at centers, velocity at edges
  - Guard cells used to set tangential velocity
- No slip condition used on solid faces
  - $\partial u / \partial n = 0$  used on open face
- Forward difference for time integration



# Water Infiltration (Diffusion only)

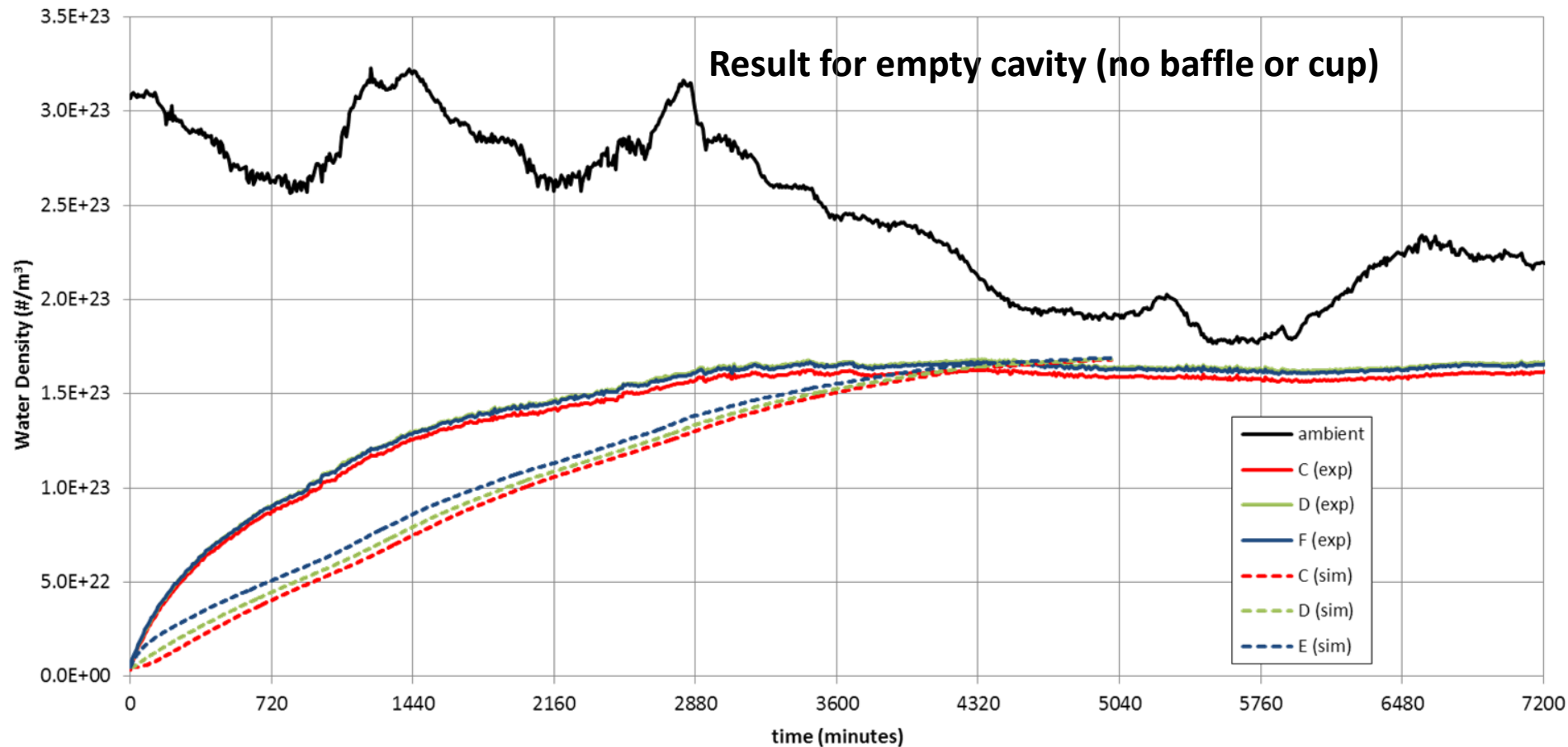
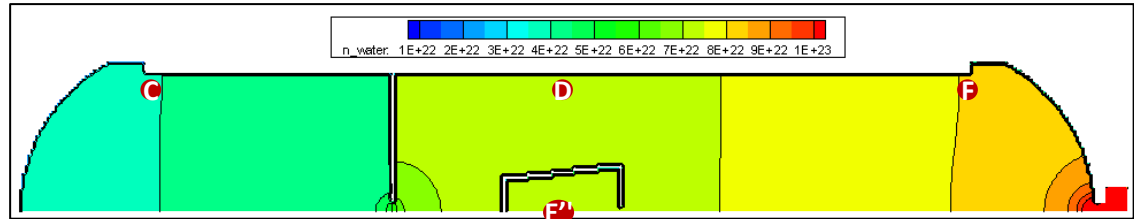


- Above animation shows water infiltration for a hypothetical test with both the baffle and the internal volume
- Plot on right is a close up of the aperture
  - Highlights typical concentration gradient
  - Also shown is the mesh resolution
- Diffusion-only cases took several hours on a workstation
  - Advection required much smaller steps to retain convergence

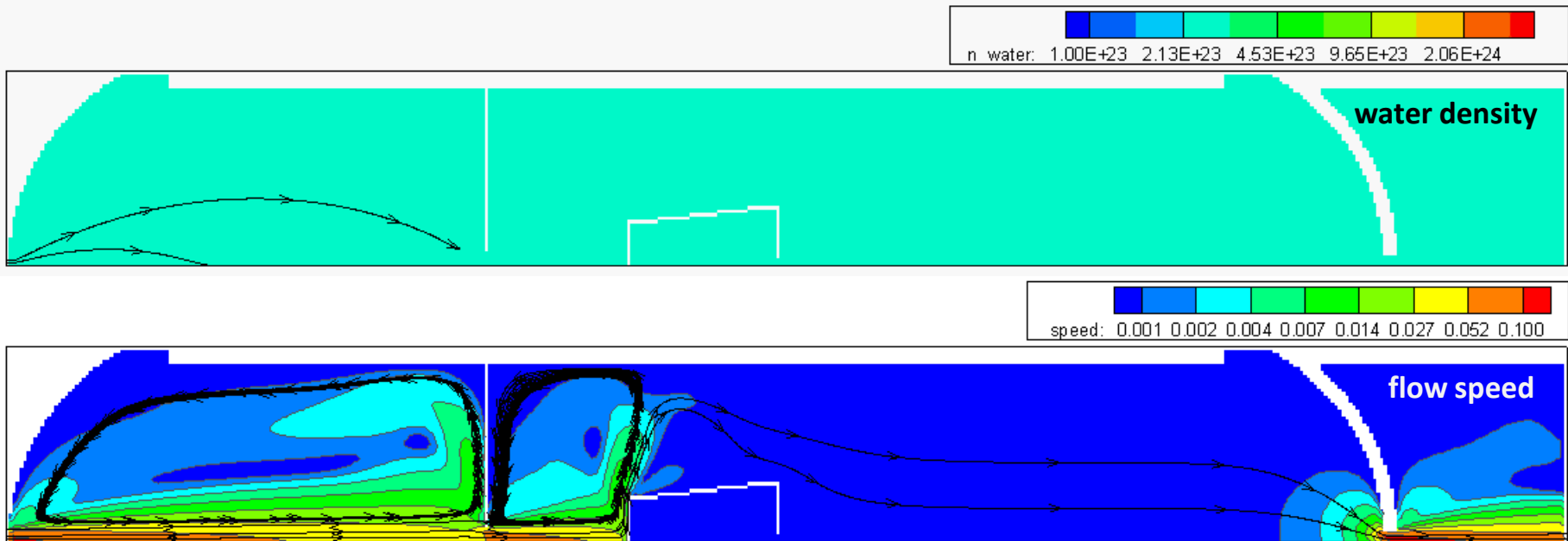


# Comparison with Experiment

- Used virtual sensors to collect pressure and humidity



# Preliminary AD+NS Results



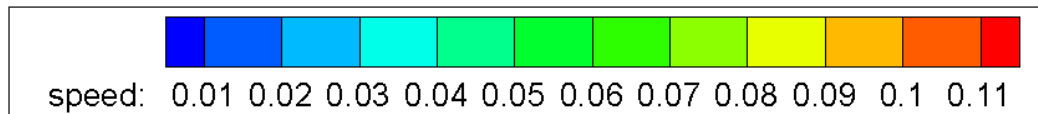
- Advection solver needed a flux limiter: numerical error or  $\Delta t$  too large?
- Velocity streamlines show complex internal flow profile
  - As predicted by Reynolds number, turbulent in the tube and laminar through aperture
  - Areas inside a vortex have a higher water concentration
  - Water from internal volume primarily removed by diffusion due to a concentration gradient

**Numerical density limiter results in non-physical mass increase: to be fixed in future work. Also, need turbulent model.**

## **PART III. IMPACT ON PARTICULATES**

# Numerical Model

- **What is the effect of purge on particulate contamination?**
  - Can use particle tracing code to investigate in details
- Simulation particles traced according to  $x^{k+1} = x^k + v^{k+0.5} \Delta t$ 
  - Particle aspect ratio and area varies randomly
- Particles injected with small random velocity
- Drag Force:  $F_d = \frac{1}{2} \rho v^2 C_d A$ 
  - Drag coefficient  $C_d = \frac{24.0}{Re} + \frac{6.0}{1+\sqrt{Re}} + 0.4$
  - Assumed ellipsoidal particles,  $A = \pi ab$
- Also gravity:  $F_g = mg$ , acting in  $-z$  direction



Particle initial position  
and **view on next page**





# Results

No flow

10Å

250Å

2000Å

- As can be verified by analysis, purge is effective in preventing infiltration of light particulates
- This particular purge not effective for heavy particulates (>2000Å)

# Conclusion

- **Performed a dual experimental and numerical effort to study purge and water infiltration**
- **Experimental Effort:**
  - The variation in internal sensors was less significant than anticipated
    - Adds credence to the simple 0D model if no internal geometry
  - Water density followed exponential decays (as predicted by the simple model), however, a double exponential resulted in a better fit
    - However, the fit parameters differed from analytical values
- **Numerical Study:**
  - The combined Advection/Diffusion and incompressible NS approach seems as a viable way to study purge and water infiltration in more detail
  - Results show generally good agreement with experiment but additional work is needed:
    - Density limiter was needed with advection term, resulted in mass increase
  - Performed particle tracing study
- **Future work: (1) turbulent model, (2) incorporate detailed surface adsorption/desorption model, (3) perform detailed study using parallel resources**

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**Questions?**